

Mobius Vibration Analysis Category-II Certification Practice Test (Sample)

Study Guide



Everything you need from our exam experts!

Copyright © 2025 by Examzify - A Kaluba Technologies Inc. product.

ALL RIGHTS RESERVED.

No part of this book may be reproduced or transferred in any form or by any means, graphic, electronic, or mechanical, including photocopying, recording, web distribution, taping, or by any information storage retrieval system, without the written permission of the author.

Notice: Examzify makes every reasonable effort to obtain from reliable sources accurate, complete, and timely information about this product.

SAMPLE

Questions

SAMPLE

- 1. What is the most common type of sensor used for vibration measurement?**
 - A. Microphone**
 - B. Thermometer**
 - C. Accelerometer**
 - D. Speed sensor**
- 2. Which of these accelerometers' sensitivities would best measure very low amplitude vibration?**
 - A. 100 mV/g**
 - B. 50 mV/g**
 - C. 10 mV/g**
 - D. 500 mV/g**
- 3. What consequence arises from not correcting a soft foot condition?**
 - A. The bearings and seals would be under stress**
 - B. The case of the motor would be distorted**
 - C. It would be difficult to align the machine**
 - D. All of these reasons and others are correct**
- 4. If the 10X peak is at 2500 CPM, what is the machine's running speed?**
 - A. 2500**
 - B. 25000**
 - C. 41.6**
 - D. 250**
- 5. Is it possible to detect bearing wear with time waveform analysis?**
 - A. No**
 - B. Yes**
 - C. Only sometimes**
 - D. Not applicable**

- 6. A periodic but non-sinusoidal wave will produce what kind of spectrum?**
- A. A single peak**
 - B. Sidebands**
 - C. An attractive spectrum**
 - D. Harmonics**
- 7. Why would an Operational Deflection Shape (ODS) test be performed?**
- A. To visualize vibration, which may include mode shapes due to resonance**
 - B. To generate an animation of the structure for your records**
 - C. To model the structure of the machine mathematically to understand its modal parameters**
 - D. To test how the machine vibration changes as speed changes**
- 8. In vibration analysis, what could a spike in the frequency spectrum indicate?**
- A. A normal operational condition**
 - B. A specific fault or defect at a corresponding frequency**
 - C. A change in lubrication quality**
 - D. An increase in machine temperature**
- 9. If a rotor has static unbalance, what is the phase relationship of the two ends of the machine in the vertical direction?**
- A. The two ends will be in phase**
 - B. The two ends will be out of phase**
 - C. There will be random phase relationships**
 - D. It depends on the load**
- 10. How many revolutions of the shaft would you ideally like to see in a time waveform if the fault condition causes vibration changes multiple times per revolution?**
- A. 10-50**
 - B. 100+**
 - C. 50-100**
 - D. 4-10**

Answers

SAMPLE

1. C
2. D
3. D
4. D
5. B
6. D
7. A
8. B
9. A
10. D

SAMPLE

Explanations

SAMPLE

1. What is the most common type of sensor used for vibration measurement?

- A. Microphone**
- B. Thermometer**
- C. Accelerometer**
- D. Speed sensor**

The most common type of sensor used for vibration measurement is the accelerometer. This device is specifically designed to measure the acceleration of a vibrating object, which is directly related to its vibrational characteristics. Accelerometers can capture both the amplitude and frequency of vibrations, providing crucial data for diagnosing equipment conditions and potential mechanical failures. The ability to measure vibrations across various frequency ranges makes accelerometers highly versatile and effective in numerous applications, particularly in industrial settings. They come in various forms, including piezoelectric, capacitive, and MEMS accelerometers, each suited to specific measurement needs. Additionally, they can be used in combination with advanced data processing techniques to enhance the accuracy and reliability of vibration analysis. In contrast, a microphone primarily measures sound pressure levels rather than vibrations, making it unsuitable for precise vibration assessment. A thermometer measures temperature, which does not relate to vibration data. A speed sensor, while relevant in certain contexts such as rotational measurement, does not provide the comprehensive vibrational information that accelerometers can offer. Therefore, accelerometers are the definitive choice for effective vibration measurement.

2. Which of these accelerometers' sensitivities would best measure very low amplitude vibration?

- A. 100 mV/g**
- B. 50 mV/g**
- C. 10 mV/g**
- D. 500 mV/g**

The best choice for measuring very low amplitude vibration is the option with the highest sensitivity, which is 500 mV/g. Sensitivity in accelerometers refers to the output voltage generated per unit of acceleration, typically measured in millivolts per gravitational unit (mV/g). A higher sensitivity indicates that the accelerometer can produce a larger output voltage for a given small vibration, making it easier to detect subtle changes in acceleration. In the context of measuring low amplitude vibrations, having an accelerometer with 500 mV/g sensitivity means that it can effectively capture smaller vibrations with greater precision. When vibrations are minimal, using a sensor with lower sensitivity would result in smaller output signals, which may be difficult to distinguish from noise, leading to potential inaccuracies in measurements. Therefore, the option with 500 mV/g will provide a much clearer and more reliable signal for analysis in applications where low amplitude vibrations are of interest.

3. What consequence arises from not correcting a soft foot condition?

- A. The bearings and seals would be under stress**
- B. The case of the motor would be distorted**
- C. It would be difficult to align the machine**
- D. All of these reasons and others are correct**

Not correcting a soft foot condition can lead to several significant issues with the machinery, impacting overall performance and longevity. When a soft foot condition is present, it indicates that one or more feet of a machine (such as a motor or pump) are not making proper contact with the base or foundation. This misalignment can create stress on the bearings and seals, leading to premature wear and potential failure. A stressed bearing may not operate efficiently, which could result in overheating and eventually result in breakdown. Additionally, the misalignment caused by a soft foot can distort the case of the motor. This distortion can lead to further issues, such as increased vibration, which can cascade into other components of the system, exacerbating the wear and tear on the machine. Correcting a soft foot condition is also imperative for maintaining proper alignment of the machine. If a soft foot is not addressed, obtaining and maintaining precise alignment becomes exceedingly difficult, potentially leading to severe operational inefficiencies, increased vibration, and further mechanical problems down the line. All of these interrelated consequences underline the importance of addressing a soft foot condition promptly to avoid a chain reaction of machinery issues, thus solidifying why it is essential to resolve the problem and maintain optimal machinery performance.

4. If the 10X peak is at 2500 CPM, what is the machine's running speed?

- A. 2500**
- B. 25000**
- C. 41.6**
- D. 250**

In vibration analysis, the concept of harmonics is essential when determining running speeds. The 10X peak indicates that the frequency measured is ten times the fundamental running speed of the machine. To find the machine's running speed, you divide the peak value by 10. In this scenario, with the 10X peak occurring at 2500 cycles per minute (CPM), the calculation would be: $2500 \text{ CPM} \div 10 = 250 \text{ CPM}$. This result indicates that the machine is operating at a fundamental frequency of 250 CPM, which correlates to its running speed. The other choices, while numerically might seem reasonable, do not appropriately represent the calculation based on the information provided regarding the 10X harmonic.

5. Is it possible to detect bearing wear with time waveform analysis?

- A. No**
- B. Yes**
- C. Only sometimes**
- D. Not applicable**

Detecting bearing wear using time waveform analysis is indeed possible and is a critical aspect of vibration analysis. Time waveform analysis involves measuring the vibration signal directly from the machine and observing the waveform produced over time. This allows for a detailed examination of the machine's mechanical behavior, including identifying characteristics that indicate wear or failure. As bearings begin to wear, they exhibit changes in their vibration patterns. These changes can manifest as alterations in the amplitude, frequency, or nature of the vibration signal. For instance, wear in the raceways or rolling elements can result in impacts or increased noise, which can be captured in the time domain. By analyzing the time waveform data, practitioners can detect abnormalities that are indicative of wear, such as amplitude spikes corresponding to defects or periodic impacts that suggest rolling element issues. This capability makes time waveform analysis a valuable tool in predictive maintenance strategies, allowing for early intervention before catastrophic failures occur. In summary, time waveform analysis is effective in detecting bearing wear as it enables the identification of subtle changes in vibration characteristics that signify degradation of bearing components.

6. A periodic but non-sinusoidal wave will produce what kind of spectrum?

- A. A single peak**
- B. Sidebands**
- C. An attractive spectrum**
- D. Harmonics**

A periodic but non-sinusoidal wave is characterized by its repetition over time, but unlike a sinusoidal wave, it does not have a simple smooth waveform. Instead, it can have sharp edges, varying amplitudes, or different shapes, leading to the generation of multiple frequency components when analyzed in the frequency domain. When performing a Fourier analysis on a periodic non-sinusoidal wave, the result reveals a harmonic spectrum, which consists of the fundamental frequency and its integer multiples—these are referred to as harmonics. This occurs because non-sinusoidal waveforms introduce additional frequency components that are based on integer multiples of the fundamental frequency due to the non-linear nature of their shapes. Thus, the presence of harmonics in the spectrum is a direct result of this complex waveform, making it the correct choice in this context. The presence of harmonics provides insight into how the waveform interacts with the physical system being observed, and helps in characterizing the different modes of vibration present.

7. Why would an Operational Deflection Shape (ODS) test be performed?

- A. To visualize vibration, which may include mode shapes due to resonance**
- B. To generate an animation of the structure for your records**
- C. To model the structure of the machine mathematically to understand its modal parameters**
- D. To test how the machine vibration changes as speed changes**

An Operational Deflection Shape (ODS) test is performed primarily to visualize vibration patterns, which can reveal mode shapes associated with resonance. The ODS provides insight into how a structure behaves in its operational environment, allowing engineers to observe the movement of various points on the machine while it is running under normal operating conditions. This visualization can indicate specific areas where vibrations are more pronounced, which may suggest a need for further investigation into the cause of the vibrations. While generating animations for records and mathematical modeling can be beneficial for understanding structures and their behaviors, they do not capture the real-time dynamics and operational interactions that an ODS test does. Moreover, while examining changes in machine vibration with speed is also critical for diagnosing equipment performance, the primary purpose of an ODS is to observe and analyze the actual vibrational response during standard operation.

8. In vibration analysis, what could a spike in the frequency spectrum indicate?

- A. A normal operational condition**
- B. A specific fault or defect at a corresponding frequency**
- C. A change in lubrication quality**
- D. An increase in machine temperature**

A spike in the frequency spectrum is indicative of a specific fault or defect within the machinery at a corresponding frequency. When analyzing vibration data, distinct frequency spikes can signal particular issues, such as imbalance, misalignment, or bearing defects, each of which has a characteristic frequency associated with it. These spikes are vital for diagnosing problems because they provide insight into the nature of the fault. For instance, if there is a spike at a frequency correlating with the rotational speed of the shaft, it may suggest an imbalance or alignment issue. Similarly, higher frequency spikes could indicate defects in rolling element bearings, as these typically generate vibrations at unique frequencies reflective of their operational characteristics. In contrast, normal operating conditions would generally not produce such pronounced spikes; rather, they would present a more continuous and stable vibration profile. Other factors like lubrication quality and machine temperature can certainly affect overall machine performance and vibration signatures but would not typically create sharp, distinct features in the frequency spectrum that clearly indicate specific faults.

9. If a rotor has static unbalance, what is the phase relationship of the two ends of the machine in the vertical direction?

- A. The two ends will be in phase**
- B. The two ends will be out of phase**
- C. There will be random phase relationships**
- D. It depends on the load**

When a rotor has static unbalance, it means that the mass is unevenly distributed relative to its axis of rotation. This condition leads to a consistent displacement pattern as the rotor spins. In a scenario of static unbalance, the two ends of the rotor will experience the same forces and thus respond in a similar manner in terms of vibration amplitude and phase. Since both ends of the machine experience the same unbalance forces due to the rotor's weight not being evenly distributed, they will oscillate together in the same phase. This means that when one end moves up, the other end also moves up, and when one end moves down, the other end does the same simultaneously. This characteristic phase relationship results from the coupled nature of the rotor's static mass distribution impacting both ends similarly, leading to synchronous motion. Options that suggest the ends are out of phase, have random phase relationships, or depend on the load do not align with the fundamental understanding of how static unbalance affects rotor dynamics. Static unbalance creates a predictable and consistent vertical lift or drop at both ends of the rotor simultaneously, which will always be in phase.

10. How many revolutions of the shaft would you ideally like to see in a time waveform if the fault condition causes vibration changes multiple times per revolution?

- A. 10-50**
- B. 100+**
- C. 50-100**
- D. 4-10**

In vibration analysis, especially when diagnosing faults that cause vibration changes multiple times per revolution, capturing enough data points is crucial for accurate analysis. Ideally, having a time waveform that represents 4-10 revolutions of the shaft provides a clear view of the recurring patterns associated with the fault condition. When a fault leads to multiple vibration changes within a single revolution, analyzing 4-10 revolutions allows for a sufficient number of cycles to observe the variations and patterns in the waveform. This range strikes a balance between having enough data to identify faults accurately and keeping the analysis manageable without overwhelming complexity. If the number of revolutions captured were lower than this range, essential details about the vibration characteristics might be missed. Conversely, capturing more than 10 revolutions could lead to excessive data which might complicate the analysis without providing substantial additional insight into the fault condition, especially if the fault exhibits clear repetitive patterns within that 4-10 revolution range. Thus, striving for 4-10 revolutions ensures that the analyst can effectively observe and interpret vibrations induced by the fault condition without unnecessarily complicating the data set.